

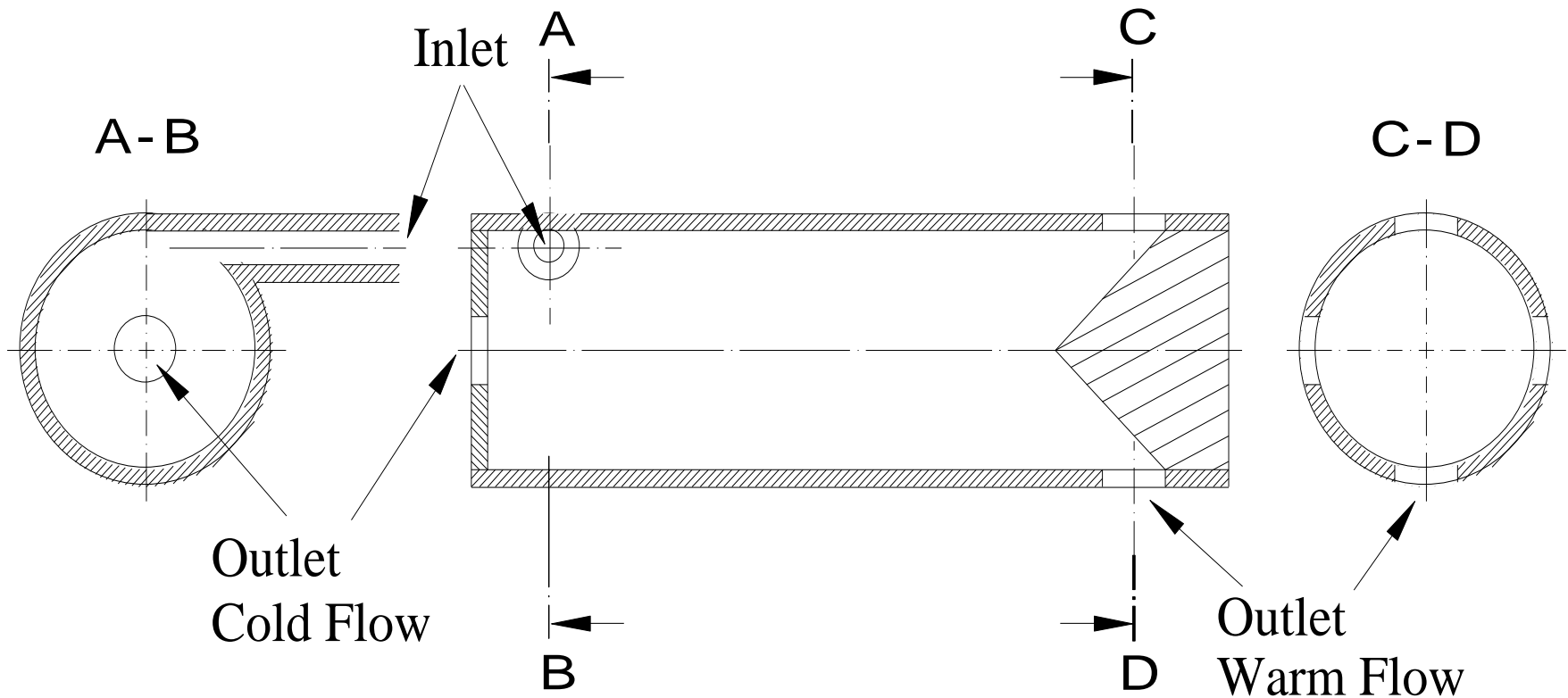
The Vortex Tube of Ranque (1931) & Hilsch (1945)

Thermodynamics and New Applications

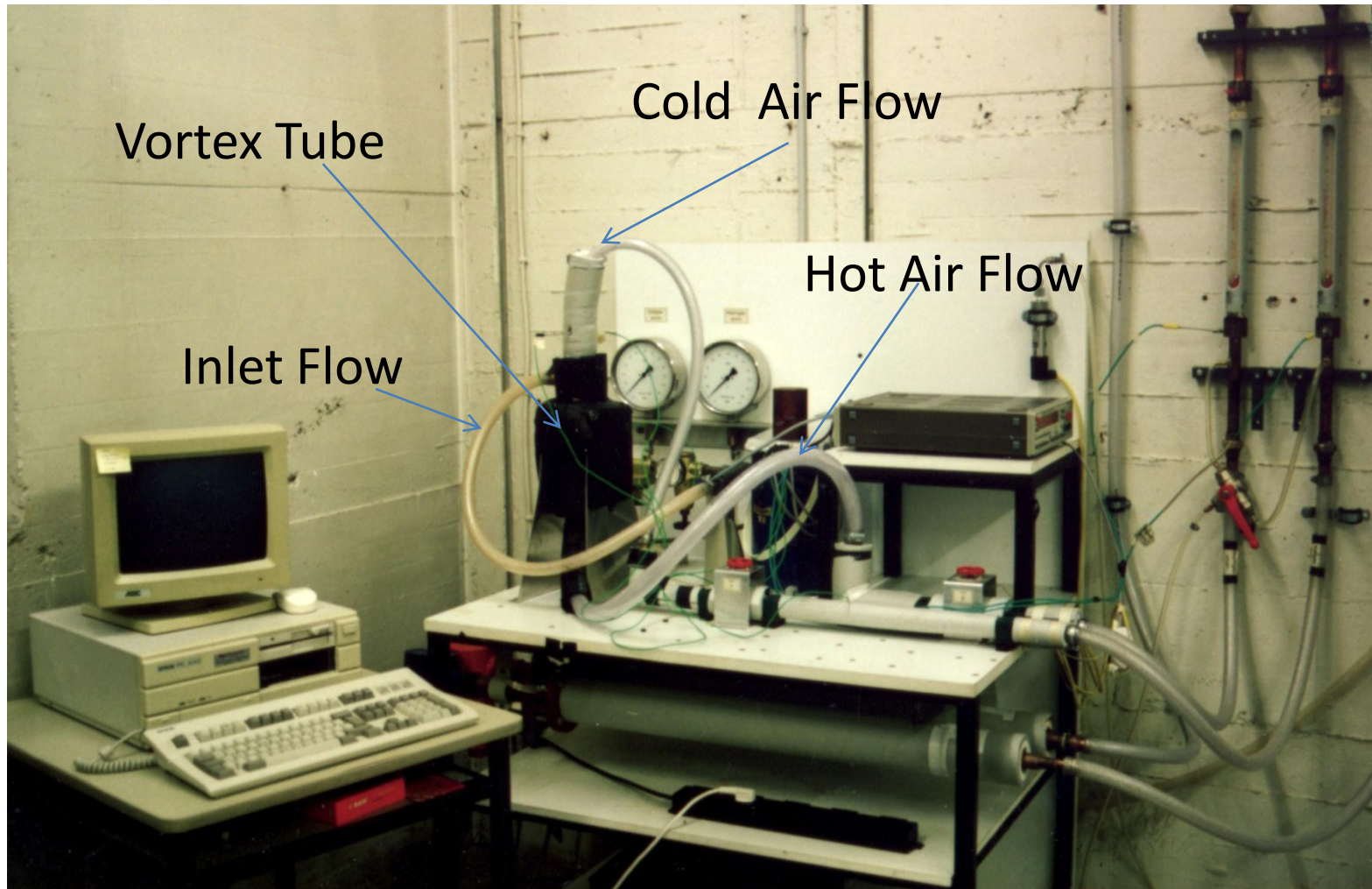
J. U. Keller, M.U. Goebel

IFT University of Siegen

keller@ift.maschinenbau.uni-siegen.de

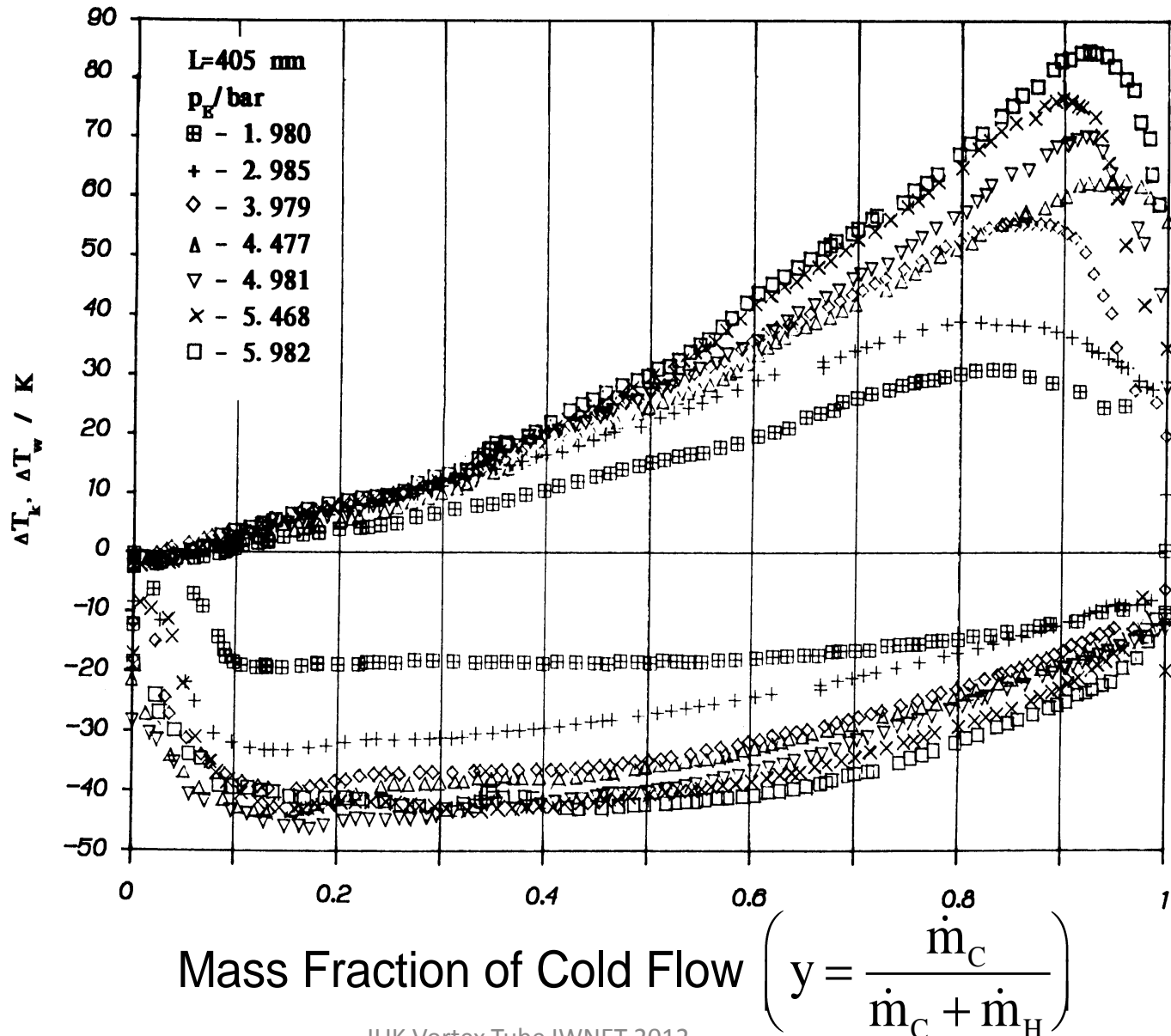


Experimental setup for expansion of compressed air in a vortex tube by Ranque & Hilsch.



Thermal Separation Effect in a Vortex Tube

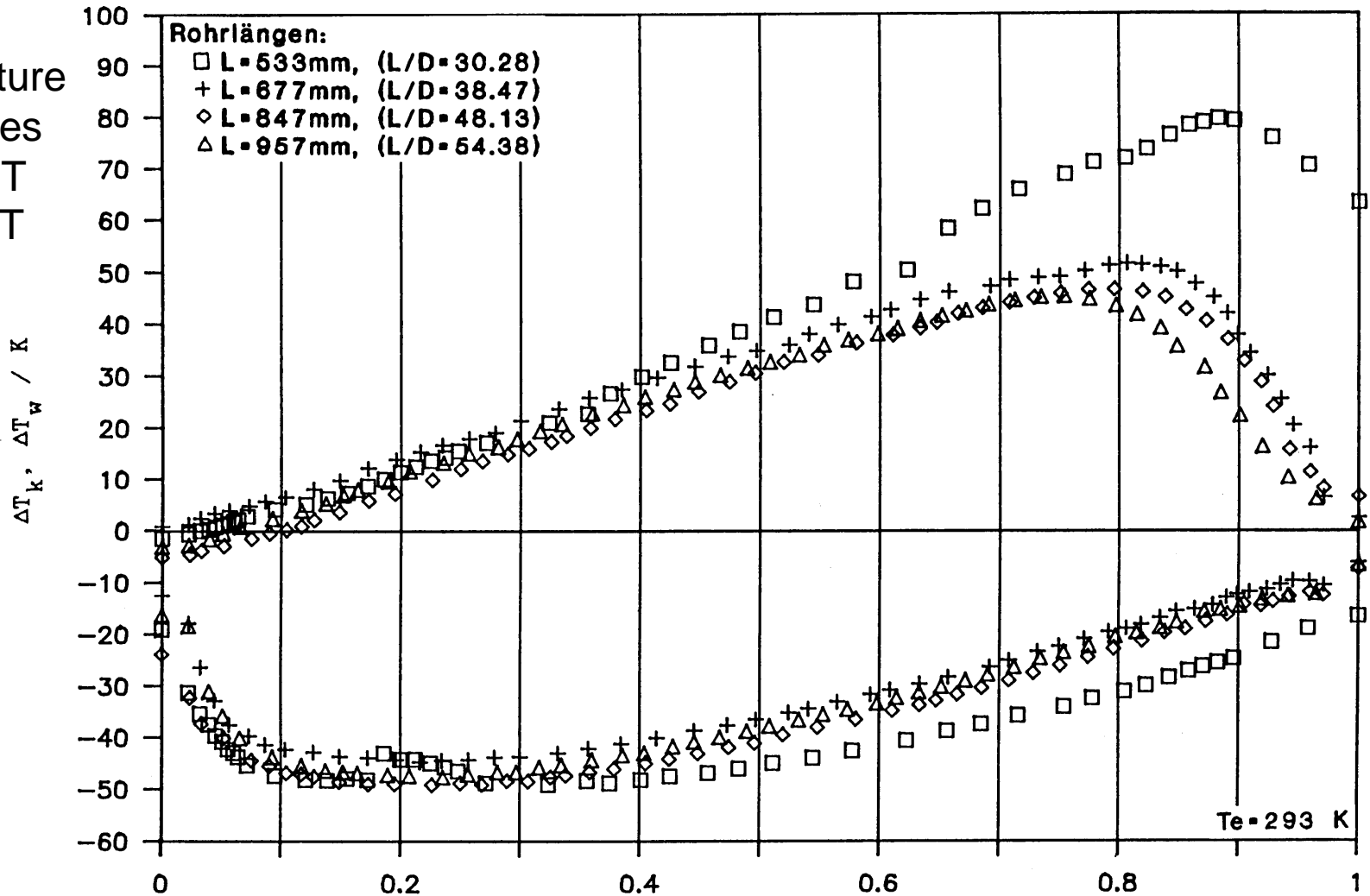
Temperature Differences
 $T(\text{hot}) - T$
 $T(\text{cold}) - T$



Thermal Separation Effect, Dependence on Tube Length

Temperature Differences
 $T(\text{hot}) - T$
 $T(\text{cold}) - T$

Temperaturdifferenzen



Massenbruch des Kaltluftstromes ($y = \dot{m}_K / (\dot{m}_K + \dot{m}_W)$)

Inlet Nozzles for Air Expansion in a Vortex Tube*

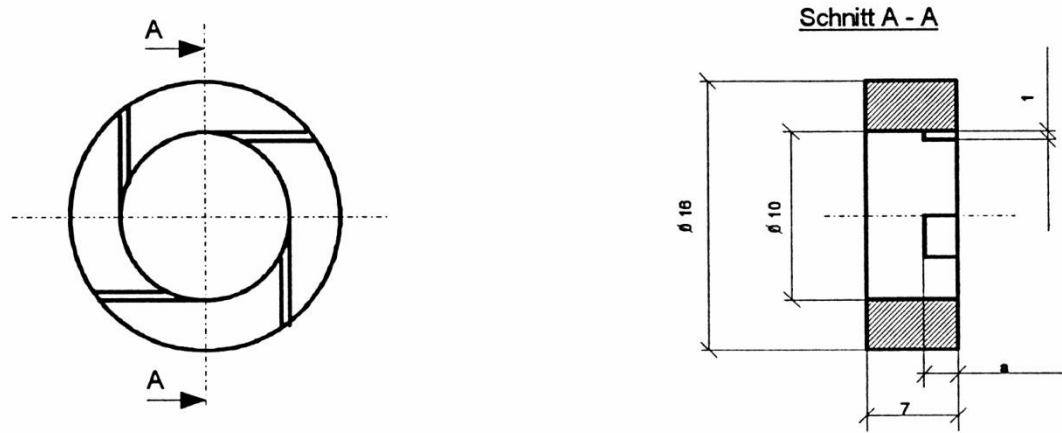
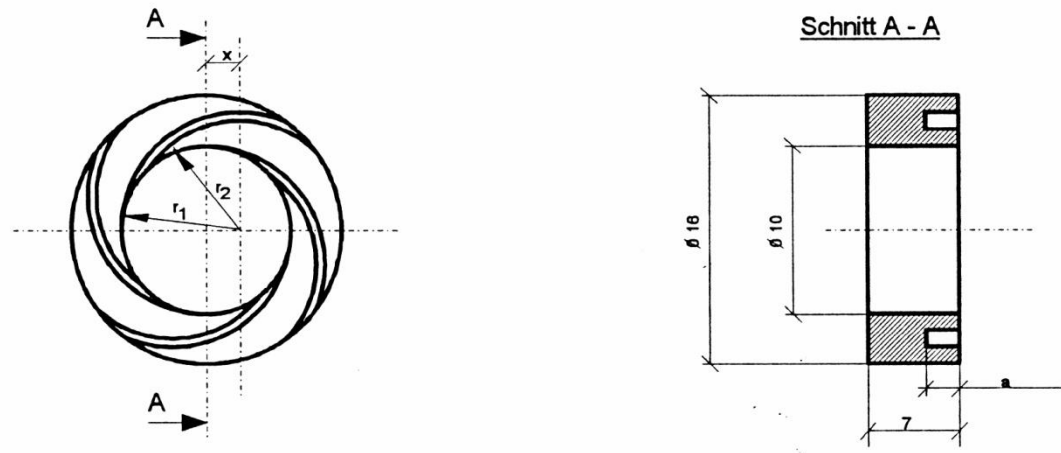


Abb.2: Düse mit geradem Einlauf und konstantem Querschnitt.

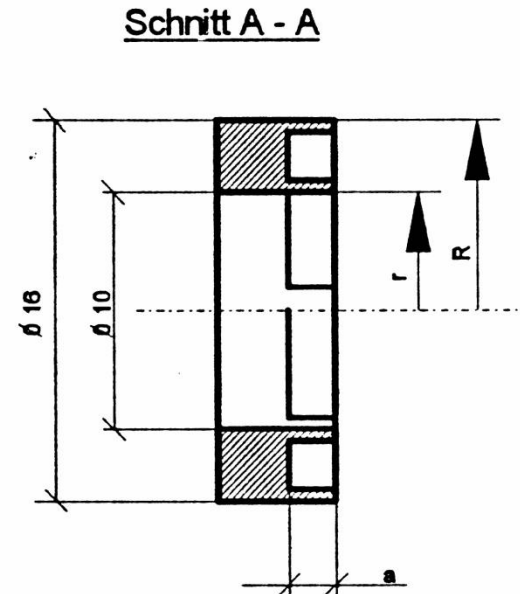
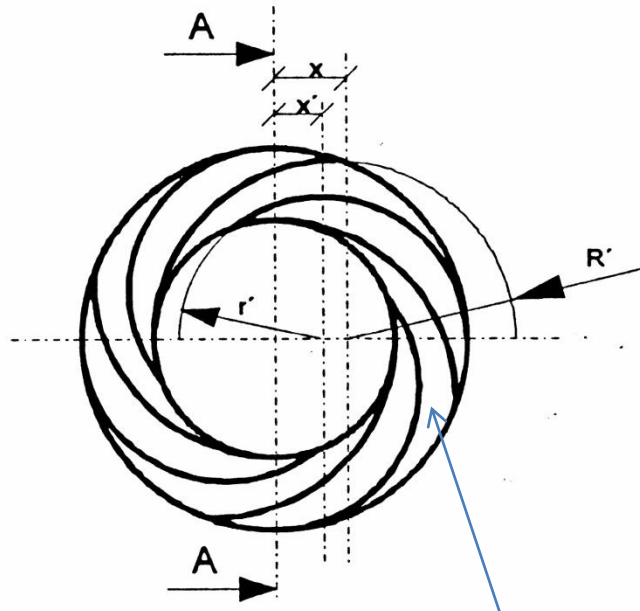
Best for
hot air flow
generation



Intermediate
performance

*Merz,H.:Experimental Investigation of the Air Expansion Process in a Vortex-Tube Using Different Types of Inlet Nozzles, IFT, U-Siegen, 1995.

Inlet Nozzles for Air Expansion in a Vortex Tube*

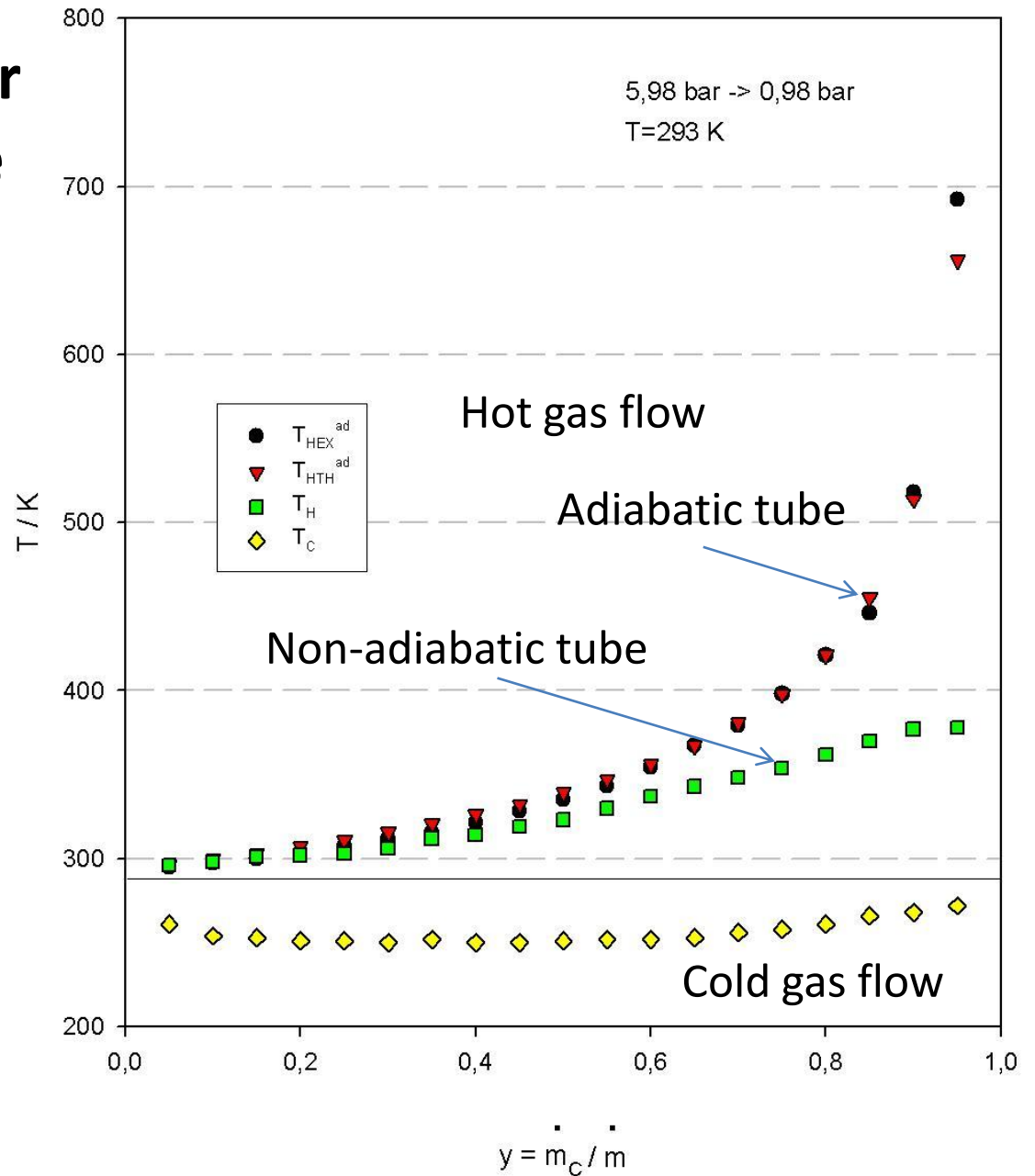


4 Spiral conic channels:
Best performance for cold air flow generation.

*Merz,H.:Experimental Investigation of the Air Expansion Process in a Vortex-Tube Using Different Types of Inlet Nozzles, IFT, U-Siegen, 1995.

Effect of Heat Transfer From the Vortex Tube

Expansion of Dry Air
 $p(0) = 5.98 \text{ bar}$
 $p^* = 0.98 \text{ bar}$



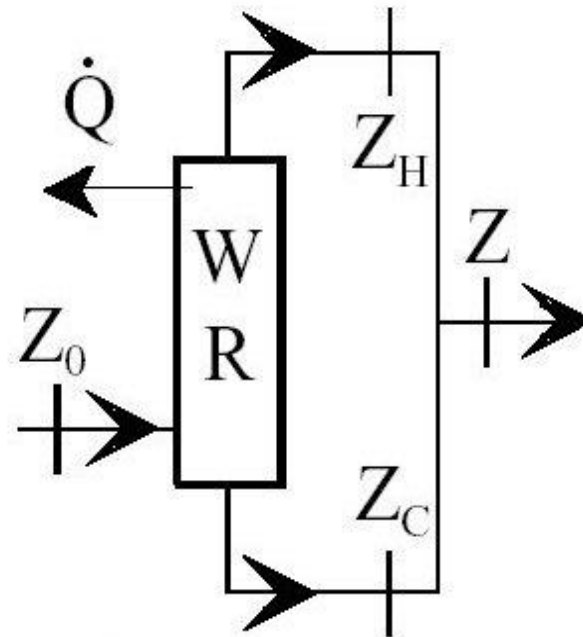
Vortex Tube Gas Expansion Process Thermodynamic Model

Mass balance

$$\dot{m}_0 = \dot{m}_C + \dot{m}_H \quad (1)$$

Mass flow

$$\dot{m}_i = A_i w_i \rho(p_i, T_i)$$



$$(1a)$$

Energy balance

$$\hat{h}_0 \dot{m}_0 = \hat{h}_C \dot{m}_C + \hat{h}_H \dot{m}_H + \dot{Q} \quad (2)$$

$$\hat{h} = h + w^2/2$$

Entropy Balance, Stationary States

.... Specific entropy of incoming fluid

.... Specific entropy of merged outgoing fluid flows

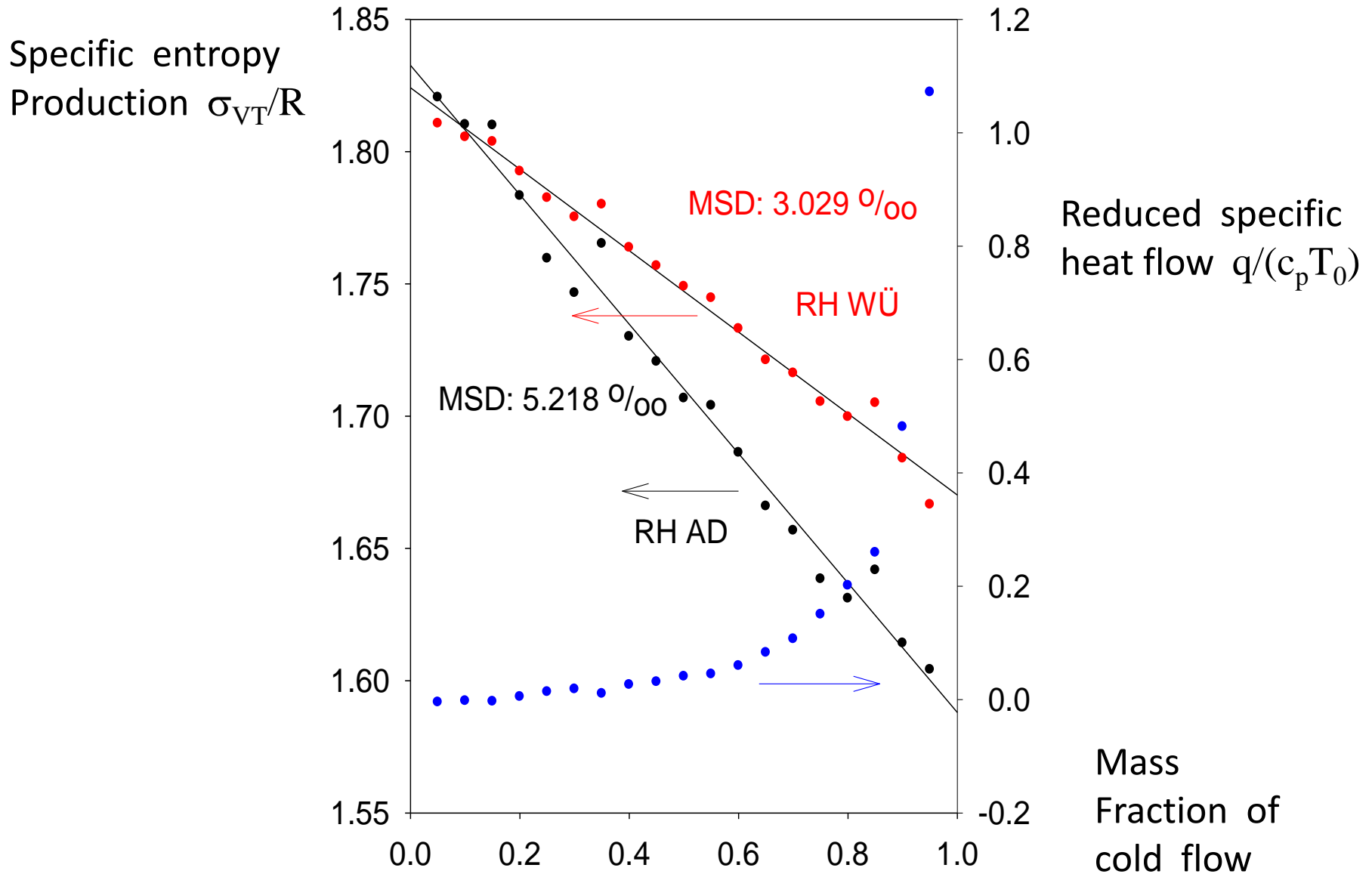
Equations of state *~ entropy of T -> s*

$$h_i = h(T_i, p_i) \quad , \quad s_i = s(T_i, p_i) \quad , \quad i = 0, C, H \quad (4,5)$$

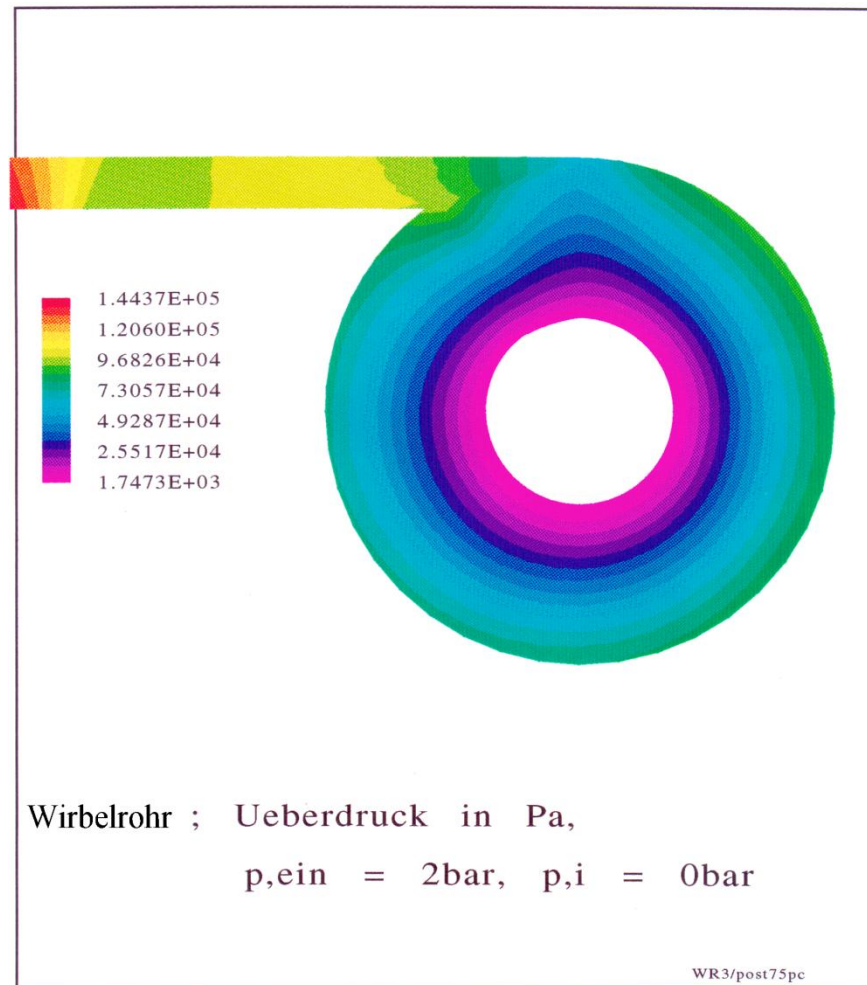
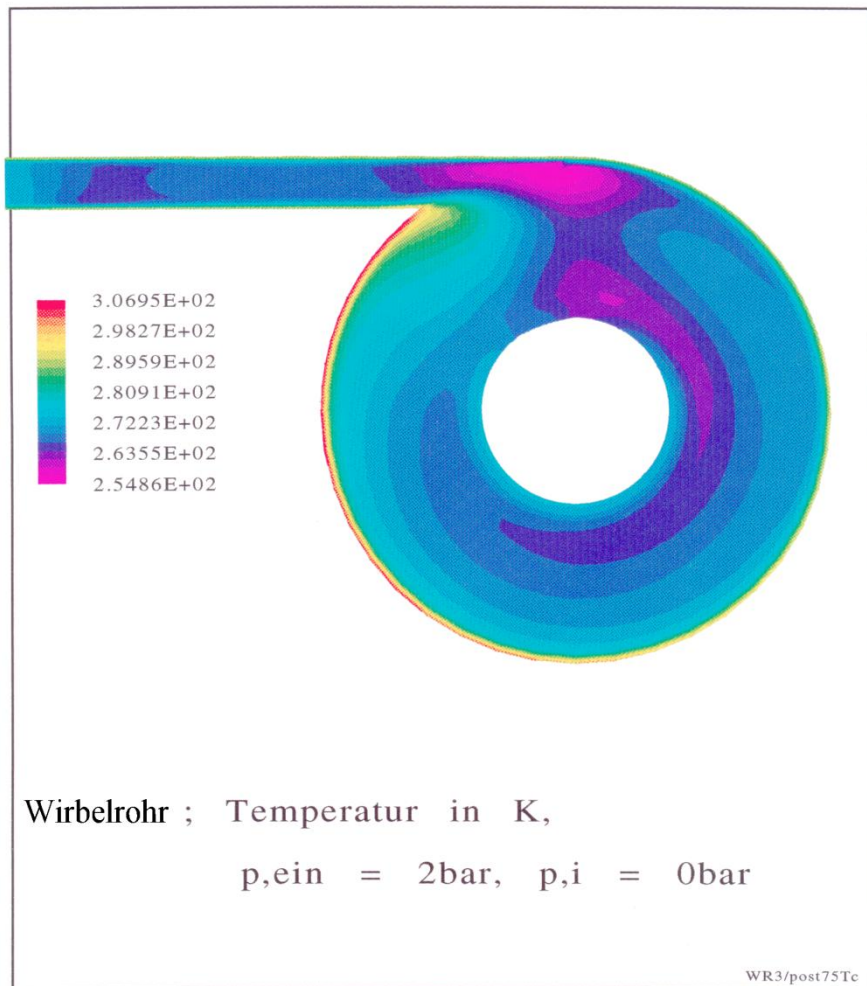
Known quantities : $m_0, m_C, T_0, p_0, p_C = p_H = p^*$

Unknown quantities : $T_C, T_H \dots (2, 3)$

Vortex Tube Process: Entropy Production, Heat Release

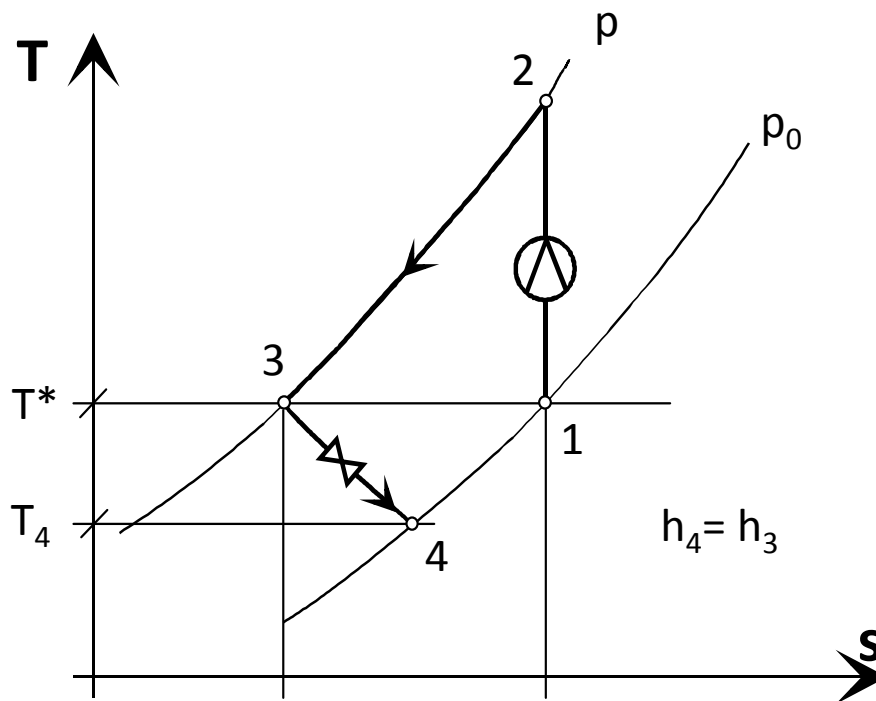
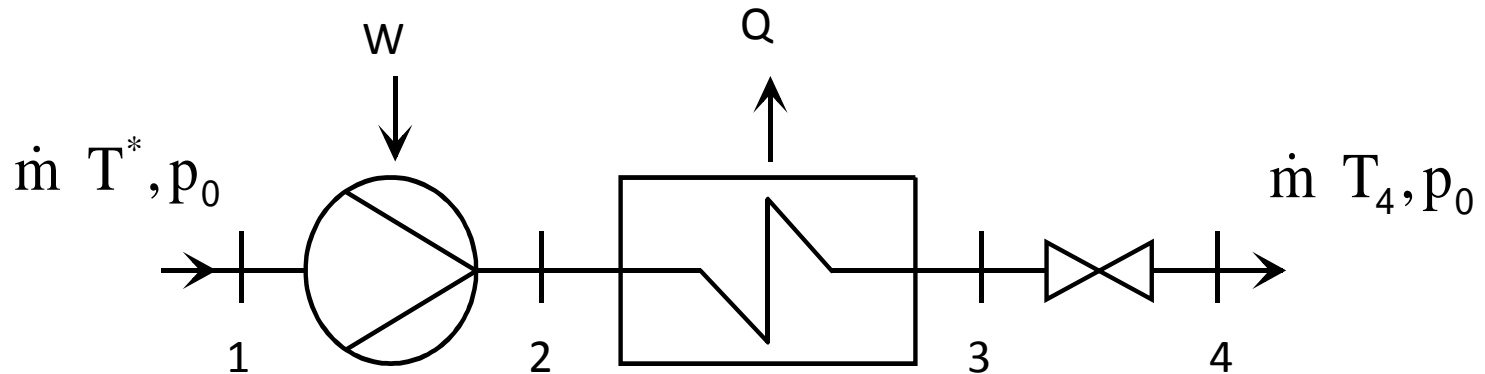


Numerical Simulation of the VT-Process



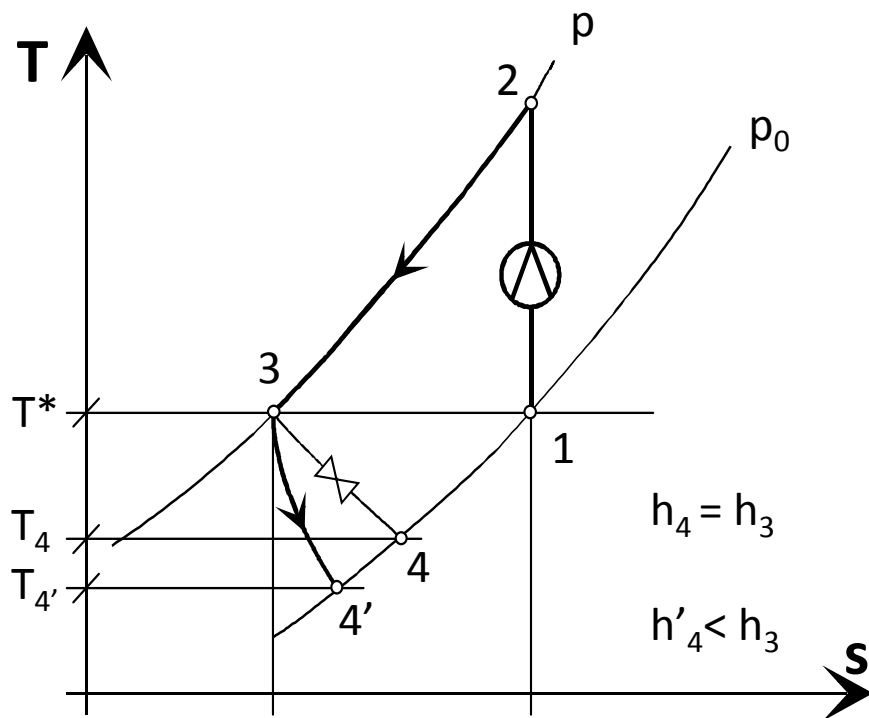
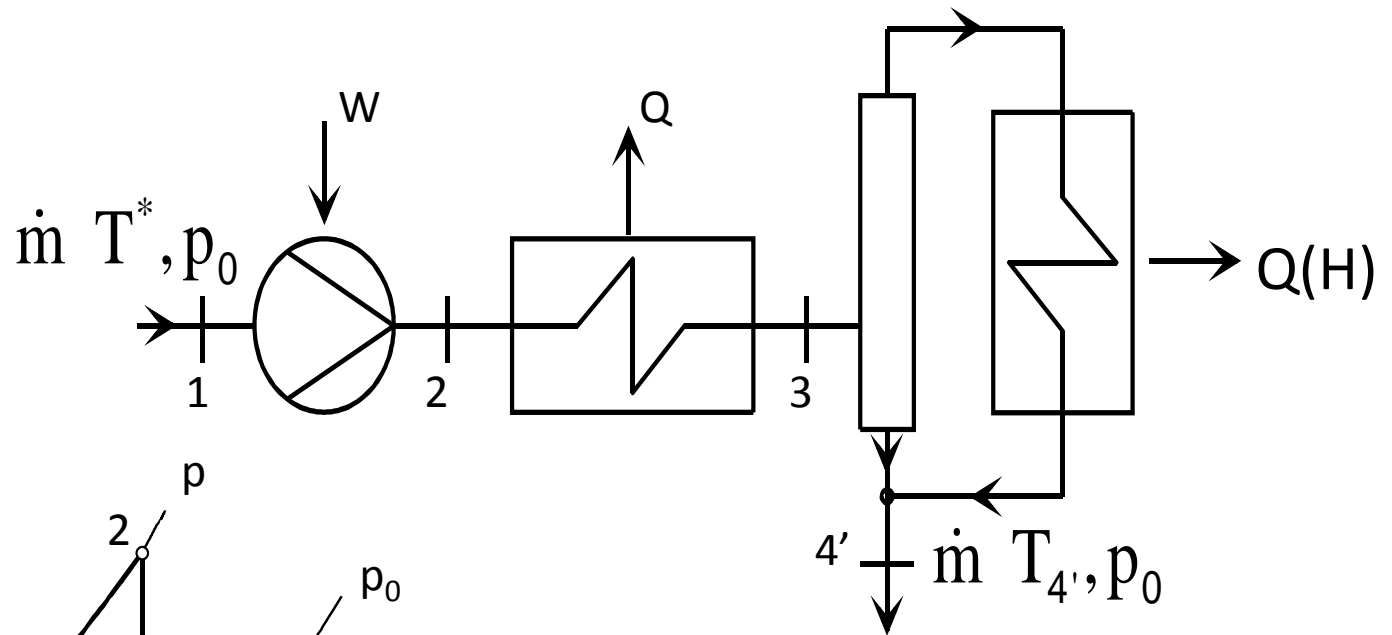
2-D Flow , Expansion of dry air. Distribution of temperature and pressure
Ref.: H. Fröhlingdorf, PHD, University of Bochum, 2001.

Claudé Gas Cooling Process



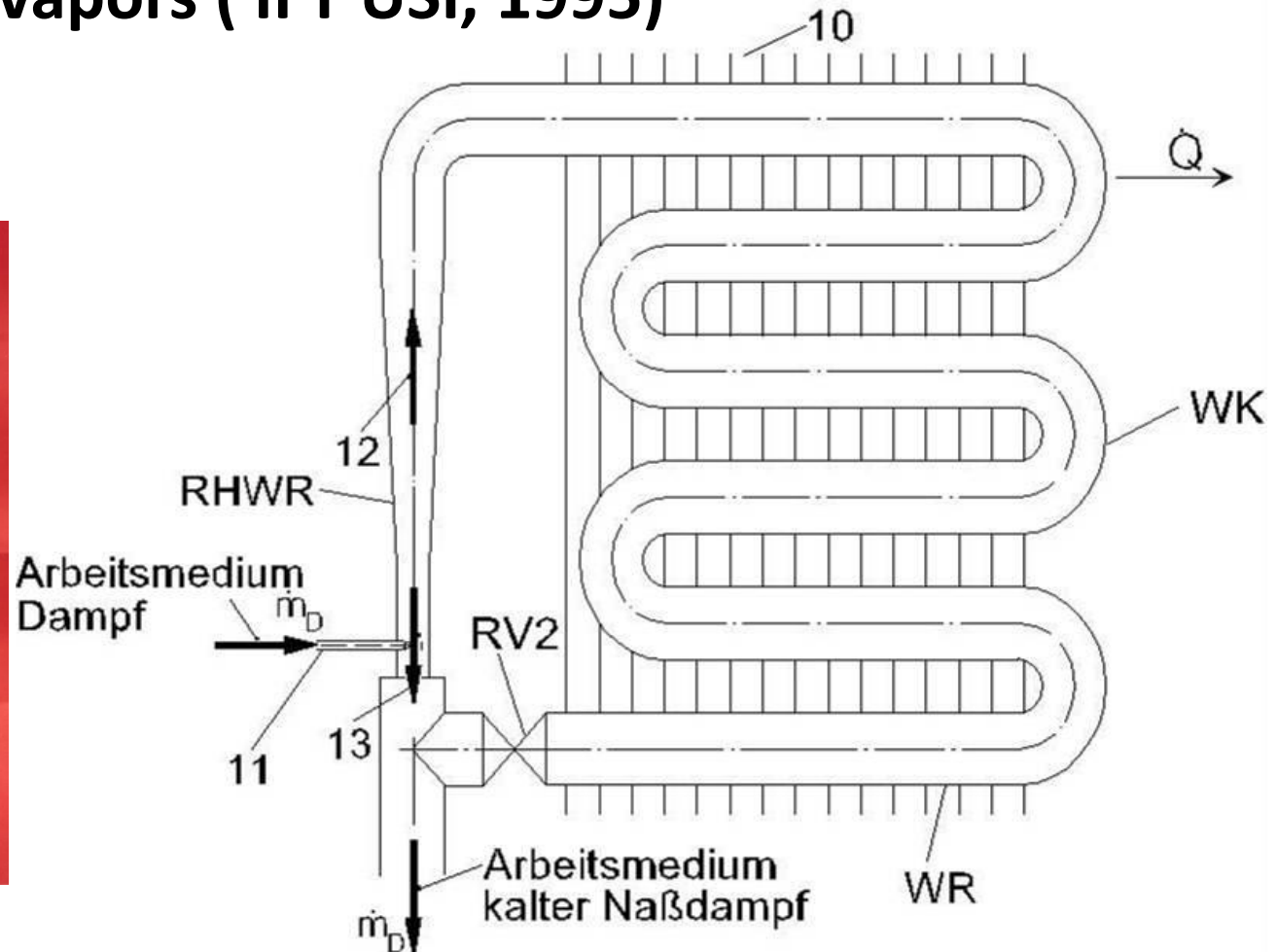
Improvement:
Substitute the
Expansion Valve
by a **Vortex Tube**

Gas Cooling Process using a Vortex Tube



Improvement:
Compression energy
savings: (3-10)%

Cooling Vortex Tube* for non-adiabatic expansion of gases and vapors (IFT USI, 1995)

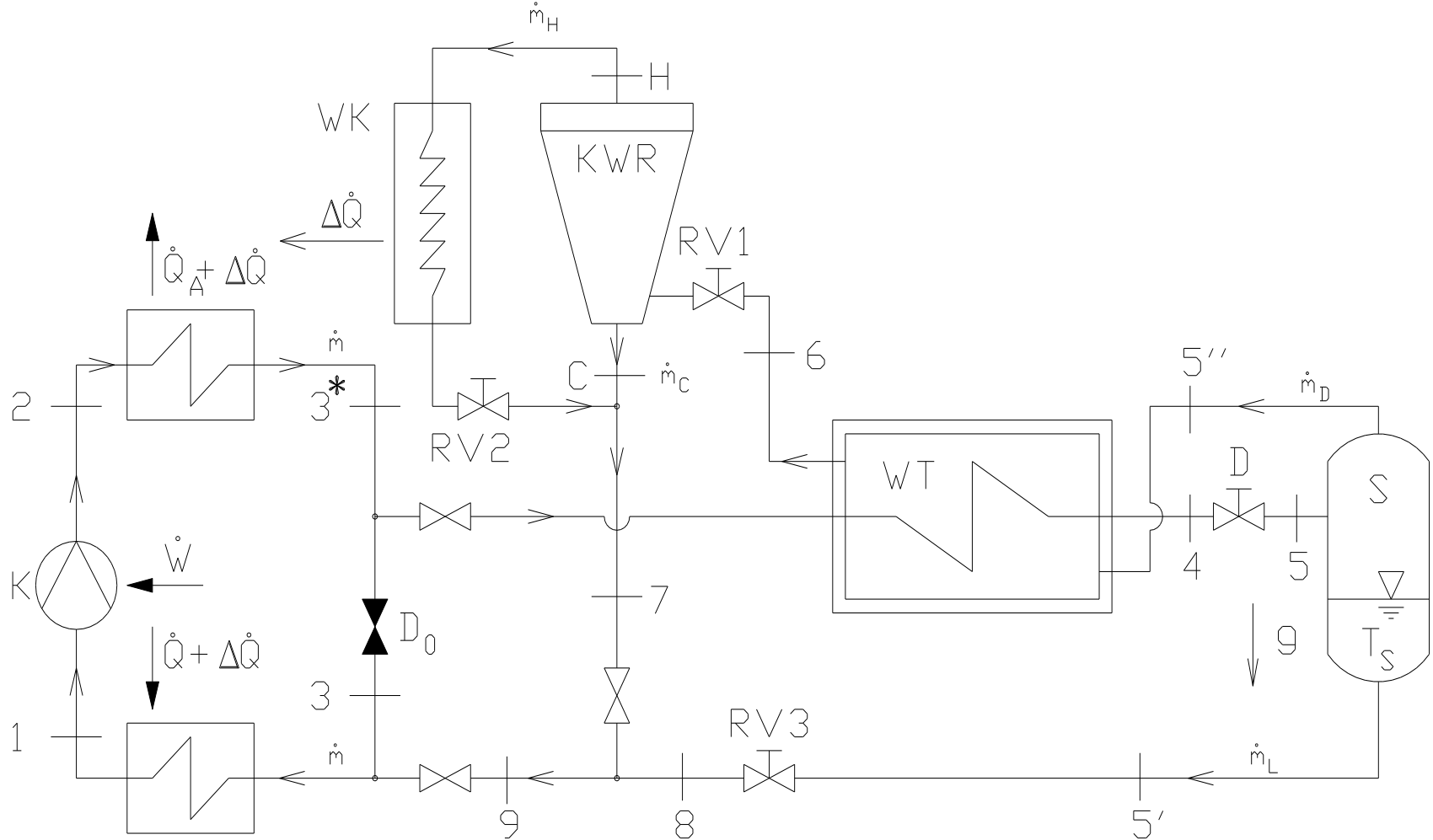


Figur 6

Kühlwirbelrohr (KWR)

*Patent DE 4345 137 A1, 1993

Non-adiabatic Expansion of Compressed Liquids

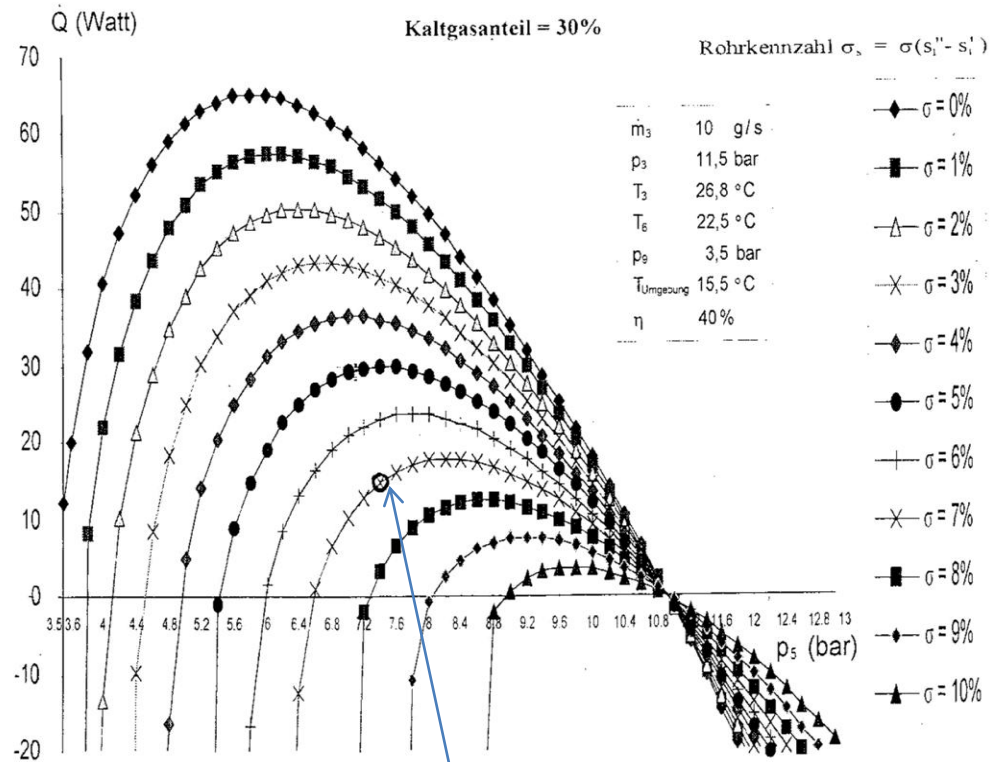
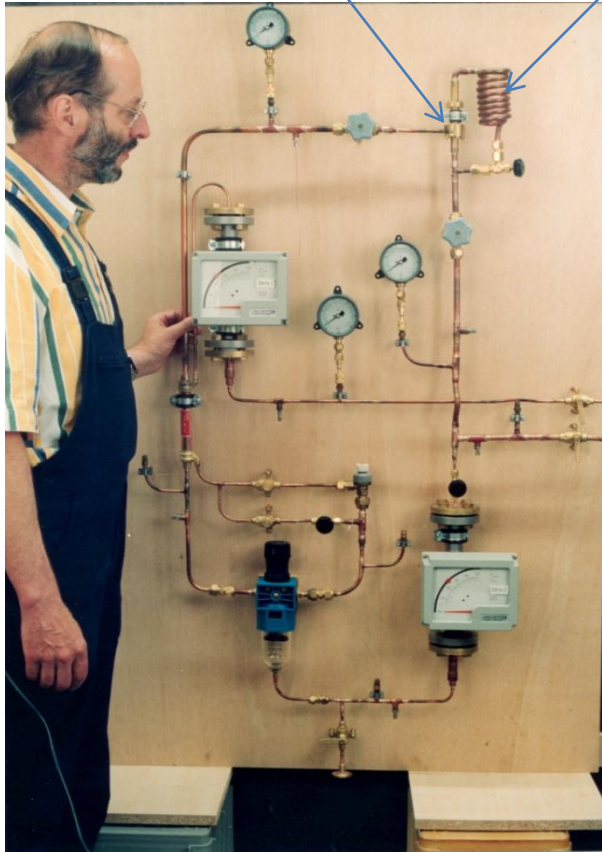


Refrigeration cycle
Expansion valve

Thermostatic expansion valve
Vortex tube
Phase separator

Thermo-Valve*: Non-adiabatic Expansion of Compressed Liquids (R22, CO2 etc.)

Vortex tube Heat exchanger



* Patent DE4343 088 A1, 1993

Measurement, R22 Refrigeration process



References:

[1] Ranque, G. J.

Experiences sur la detente giratoire avec production simultanes d'un echappement d'air chaud et d'un echappement d'air froid,
Journal de physique et le radium, 4 (1933) ; No. 7.

[2]Hilsch, R.

Die Expansion von Gasen im Zentrifugalfeld als Kälteprozess,
Zeitschrift für Naturforschung, 1 (1946), 208-214.

[3]Schäfer, M.

Untersuchung von Entspannungsvorgängen komprimierter Luft am Wirbelrohr nach Ranque und Hilsch, Studienarbeit,
Lehrstuhl für Thermodynamik, IFT, Universität Siegen, Siegen 1989.

[4]Plank, Rudolf. (Hsg.)

Handbuch der Kältetechnik, Bd. III (XII),
Verfahren der Kälteerzeugung, Kap. 3, p. 18 ff., Springer, Berlin, 1961.

<http://www.uni-siegen.de/fb11/tts/personen/juk/?lang=de>